# Benchmark Computations of Laminar Flow Around a Cylinder 

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#### Abstract

SUMMARY An overview of benchmark computations for 2D and 3D laminar flows around a cylinder is given, which have been defined for a comparison of different solution approaches for the incompressible Navier-Stokes equations developed within the Priority Research Programme. The exact definitions of the benchmarks are recapitulated and the numerical schemes and computers employed by the various participating groups are summarized. A detailed evaluation of the results provided is given, also including a comparison with a reference experiment. The principal purpose of the benchmarks is discussed and some general conclusions which can be drawn from the results are formulated.


## 1. INTRODUCTION

Under the DFG Priority Research Program "Flow Simulation on High Performance Computers" solution methods for various flow problems have been developed with considerable success. In many cases, the computing times are still very long and, because of a lack of storage capacity and insufficient resolution, the agreement between the computed results and experimental data is - even for laminar flows - only qualitative in nature. If numerical solutions are to play a similar role to wind tunnels, they have to provide the same accuracy as measurements, in particular in the prediction of the overall forces.

Several new techniques such as "unstructured grids", "multigrid", "operator splitting", "domain decomposition" and "mesh adaptation" have been used in order to improve the performance of numerical methods. To facilitate the comparison of these solution approaches, a set of benchmark problems has been defined and all participants of the Priority Research Program working on incompressible flows have been invited to submit their solutions. This paper presents the results of these computations contributed by altogether 17 research groups, 10 from within of the Priority Research Program and 7 from outside. The major purpose of the benchmark is to establish, whether constructive conclusions can be drawn from a comparison of these results so that the solutions can be improved. It is not the aim to come to the conclusion that a particular solution A is better than another solution $B$; the intention is rather to determine whether and why certain
approaches are superior to others. The benchmark is particularly meant to stimulate future work.

In the first step, only incompressible laminar test cases in two and three dimensions have been selected which are not too complicated, but still contain most difficulties representative of industrial flows in this regime. In particular, characteristic quantities such as drag and lift coefficients have to be computed in order to measure the ability to produce quantitatively accurate results. This benchmark aims to develop objective criteria for the evaluation of the different algorithmic approaches. For this purpose, the participants have been asked to submit a fairly complete account of their computational results together with detailed information about the discretization and solution methods used. As a result it should be possible, at least for this particular class of flows, to distinguish between "efficient" and "less efficient" solution approaches. After this benchmark has been proved to be successful it will be extended to include also certain turbulent and compressible flows.

It is particularly hoped that this benchmark will provide the basis for reaching decisive answers to the following questions which are currently the subject of controversial discussion:

1. Is it possible to calculate incompressible (laminar) flows accurately and efficiently by methods based on explicitly advancing momentum?
2. Can one construct an efficient solver for incompressible flow without employing multigrid components, at least for the pressure Poisson equation?
3. Do conventional finite difference methods have advantages over new finite element or finite volume techniques?
4. Can steady-state solutions be efficiently computed by pseudo-time-stepping techniques?
5. Is a low-order treatment of the convective term competitive, possibly for smaller Reynolds numbers?
6. What is the "best" strategy for time stepping: fully coupled iteration or operator splitting (pressure correction scheme)?
7. Does it pay to use higher order discretizations in space or time?
8. What is the potential of using unstructured grids?
9. What is the potential of a posteriori grid adaptation and time step selection in flow computations?
10. What is the "best" approach to handle the nonlinearity: quasi-Newton iteration or nonlinear multigrid?

These questions appear to be of vital importance in the construction of efficient and reliable solvers, particularly in three space dimensions. Everybody who is extensively consuming computer resources for numerical flow simulation should be interested.

The authors have tried their best in presenting and evaluating the contributed results in as much detail as possible and hope that all participants of the benchmark will find themselves correctly quoted.

## 2. DEFINITION OF TEST CASES

This section gives a brief summary of the definitions of the test cases for the benchmark computations, including precise definitions of the quantities which had to be computed and also some additional instructions which were given to the participants.

### 2.1 Fluid Properties

The fluid properties are identical for all test cases. An incompressible Newtonian fluid is considered for which the conservation equations of mass and momentum are

$$
\begin{gathered}
\frac{\partial U_{i}}{\partial x_{i}}=0 \\
\rho \frac{\partial U_{i}}{\partial t}+\rho \frac{\partial}{\partial x_{j}}\left(U_{j} U_{i}\right)=\rho \nu \frac{\partial}{\partial x_{j}}\left(\frac{\partial U_{i}}{\partial x_{j}}+\frac{\partial U_{j}}{\partial x_{i}}\right)-\frac{\partial P}{\partial x_{i}}
\end{gathered}
$$

The notations are time $t$, cartesian coordinates $\left(x_{1}, x_{2}, x_{3}\right)=(x, y, z)$, pressure $P$ and velocity components $\left(U_{1}, U_{2}, U_{3}\right)=(U, V, W)$. The kinematic viscosity is defined as $\nu=$ $10^{-3} \mathrm{~m}^{2} / \mathrm{s}$, and the fluid density is $\rho=1.0 \mathrm{~kg} / \mathrm{m}^{3}$.

### 2.2 2D Cases

For the 2D test cases the flow around a cylinder with circular cross-section is considered. The geometry and the boundary conditions are indicated in Fig. 1. For all test cases the


Figure 1: Geometry of 2D test cases with boundary conditions
outflow condition can be chosen by the user.
Some definitions are introduced to specify the values which have to be computed. $H=$ 0.41 m is the channel height and $D=0.1 \mathrm{~m}$ is the cylinder diameter. The Reynolds number is defined by $R e=\bar{U} D / \nu$ with the mean velocity $\bar{U}(t)=2 U(0, H / 2, t) / 3$. The drag
and lift forces are

$$
F_{D}=\int_{S}\left(\rho \nu \frac{\partial v_{t}}{\partial n} n_{y}-P n_{x}\right) d S \quad, \quad F_{L}=-\int_{S}\left(\rho \nu \frac{\partial v_{t}}{\partial n} n_{x}+P n_{y}\right) d S
$$

with the following notations: circle $S$, normal vector $n$ on $S$ with $x$-component $n_{x}$ and $y$-component $n_{y}$, tangential velocity $v_{t}$ on $S$ and tangent vector $t=\left(n_{y},-n_{x}\right)$. The drag and lift coefficients are

$$
c_{D}=\frac{2 F_{w}}{\rho \bar{U}^{2} D} \quad, \quad c_{L}=\frac{2 F_{a}}{\rho \bar{U}^{2} D}
$$

The Strouhal number is defined as $S t=D f / \bar{U}$, where $f$ is the frequency of separation. The length of recirculation is $L_{a}=x_{r}-x_{e}$, where $x_{e}=0.25$ is the $x$-coordinate of the end of the cylinder and $x_{r}$ is the $x$-coordinate of the end of the recirculation area. As a further reference value the pressure difference $\Delta P=\Delta P(t)=P\left(x_{a}, y_{a}, t\right)-P\left(x_{e}, y_{e}, t\right)$ is defined, with the front and end point of the cylinder $\left(x_{a}, y_{a}\right)=(0.15,0.2)$ and $\left(x_{e}, y_{e}\right)=(0.25,0.2)$, respectively.

## a) Test case 2D-1 (steady):

The inflow condition is

$$
U(0, y)=4 U_{m} y(H-y) / H^{2}, V=0
$$

with $U_{m}=0.3 \mathrm{~m} / \mathrm{s}$, yielding the Reynolds number $R e=20$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$, length of recirculation zone $L_{a}$ and pressure difference $\Delta P$.

## b) Test case 2D-2 (unsteady):

The inflow condition is

$$
U(0, y, t)=4 U_{m} y(H-y) / H^{2}, V=0
$$

with $U_{m}=1.5 \mathrm{~m} / \mathrm{s}$, yielding the Reynolds number $R e=100$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$ and pressure difference $\Delta P$ as functions of time for one period $\left[t_{0}, t_{0}+1 / f\right]$ (with $f=f\left(c_{L}\right)$ ), maximum drag coefficient $c_{D \max }$, maximum lift coefficient $c_{L \max }$, Strouhal number $S t$ and pressure difference $\Delta P(t)$ at $t=t_{0}+1 / 2 f$. The initial data $\left(t=t_{0}\right)$ should correspond to the flow state with $c_{L \max }$.

## c) Test case 2D-3 (unsteady):

The inflow condition is

$$
U(0, y, t)=4 U_{m} y(H-y) \sin (\pi t / 8) / H^{2}, V=0
$$

with $U_{m}=1.5 \mathrm{~m} / \mathrm{s}$, and the time interval is $0 \leq t \leq 8 \mathrm{~s}$. This gives a time varying Reynolds number between $0 \leq \operatorname{Re}(t) \leq 100$. The initial data $(t=0)$ are $U=V=P=0$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$ and pressure difference $\Delta P$ as functions of time for $0 \leq t \leq 8 \mathrm{~s}$, maximum drag coefficient $c_{D \max }$, maximum lift coefficient $c_{L \max }$, pressure difference $\Delta P(t)$ at $t=8 \mathrm{~s}$.

### 2.3 3D Cases

For the 3D test cases the flows around a cylinder with square and circular cross-sections are considered. The problem configurations and boundary conditions are illustrated in Figs. 2 and 3. The outflow condition can be selected by the user. Some definitions are


Figure 2: Configuration and boundary conditions for flow around a cylinder with square cross-section.
introduced to specify the values which have to be computed. The height and width of the channel is $H=0.41 \mathrm{~m}$, and the side length and diameter of the cylinder are $D=0.1 \mathrm{~m}$. The characteristic velocity is $\bar{U}(t)=4 U(0, H / 2, H / 2, t) / 9$, and the Reynolds number is defined by $R e=\bar{U} D / \nu$. The drag and lift forces are

$$
F_{D}=\int_{S}\left(\rho \nu \frac{\partial v_{t}}{\partial n} n_{y}-p n_{x}\right) d S \quad, \quad F_{L}=-\int_{S}\left(\rho \nu \frac{\partial v_{t}}{\partial n} n_{x}+P n_{y}\right) d S
$$

with the following notations: surface of cylinder $S$, normal vector $n$ on $S$ with $x$-component $n_{x}$ and $y$-component $n_{y}$, tangential velocity $v_{t}$ on $S$ and tangent vector $t=\left(n_{y},-n_{x}, 0\right)$. The drag and lift coefficients are

$$
c_{D}=\frac{2 F_{w}}{\rho \bar{U}^{2} D H} \quad, \quad c_{L}=\frac{2 F_{a}}{\rho \bar{U}^{2} D H}
$$

The Strouhal number is $S t=D f / \bar{U}$ with the frequency of separation $f$, and a pressure difference is defined by $\Delta P=\Delta P(t)=P\left(x_{a}, y_{a}, z_{a}, t\right)-P\left(x_{e}, y_{e}, z_{e}, t\right)$ with coordinates $\left(x_{a}, y_{a}, z_{a}\right)=(0.45,0.20,0.205)$ and $\left(x_{e}, y_{e}, z_{e}\right)=(0.55,0.20,0.205)$.


Figure 3: Configuration and boundary conditions for flow around a cylinder with circular cross-section.

## a) Test cases 3D-1Q and 3D-1Z (steady):

The inflow condition is

$$
U(0, y, z)=16 U_{m} y z(H-y)(H-z) / H^{4}, V=W=0
$$

with $U_{m}=0.45 \mathrm{~m} / \mathrm{s}$, yielding the Reynolds number $R e=20$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$ and pressure difference $\Delta P$.

## b) Test cases 3D-2Q and 3D-2Z (unsteady):

The inflow condition is

$$
U(0, y, z, t)=16 U_{m} y z(H-y)(H-z) / H^{4}, V=W=0
$$

with $U_{m}=2.25 \mathrm{~m} / \mathrm{s}$, yielding the Reynolds number $R e=100$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$ and pressure difference $\Delta P$ as functions of time for three periods $\left[t_{0}, t_{0}+3 / f\right]$ (with $f=f\left(c_{L}\right)$ ), maximum drag coefficient $c_{D \max }$, maximum lift coefficient $c_{L \max }$ and Strouhal number St. The initial data $\left(t=t_{0}\right)$ are arbitrary, however, for fully developed flow.

## c) Test cases 3D-3Q and 3D-3Z (unsteady):

The inflow condition is

$$
U(0, y, z, t)=16 U_{m} y z(H-y)(H-z) \sin (\pi t / 8) / H^{4}, V=W=0
$$

with $U_{m}=2.25 \mathrm{~m} / \mathrm{s}$. The time interval is $0 \leq t \leq 8 \mathrm{~s}$. This yields a time-varying Reynolds number between $0 \leq \operatorname{Re}(t) \leq 100$. The initial data $(t=0)$ are $U=V=P=0$. The following quantities should be computed: drag coefficient $c_{D}$, lift coefficient $c_{L}$ and pressure difference $\Delta P$ as functions of time for $0 \leq t \leq 8 \mathrm{~s}$, maximum drag coefficient $c_{D \max }$, maximum lift coefficient $c_{L \max }$ and pressure difference $\Delta P(t)$ for $t=8 \mathrm{~s}$.

### 2.4 Instructions for Computations

The following additional instructions concerning the computations were given to the participants:

- In the case of the steady calculations 2D-1, 3D-1Q and 3D-1Z, the results have to be presented for three successively coarsened meshes (notation: $h_{1}, h_{2}$ and $h_{3}$ with finest level $h_{1}$ ).
- Any iterative process used for the steady computations should start from zero values.
- In the case of the unsteady calculations 2D-2, 2D-3, 3D-2Q, 3D-2Z, 3D-3Q and 3D3Z, the results have to be presented for three successively coarsened meshes (notation as in the steady case) with a finest time discretization (notation: $\Delta t_{1}$ ) and also for two successively coarsened time discretizations (notation: $\Delta t_{2}$ and $\Delta t_{3}$ ) together with the finest mesh $h_{1}$.
- The finest spatial mesh $h_{1}$, the finest time discretization $\Delta t_{1}$ and the coarsening strategies can be chosen by the user.
- The convergence criteria for the iterative method in the steady case and for each time step in the unsteady cases (in connection with implicit methods) can be chosen by the user.
- The outflow condition can be chosen by the user.
- If possible, the calculations should be performed on a workstation. For all computers used, the theoretical peak performance and the MFlop rate for the LINPACK1000benchmark (in 64 -bit arithmetic) should be provided. The LINPACK1000 value should be obtained with the same compiler options as used for the flow solver.
- In addition to the benchmark results a description of the solution methods should be given.


## 3. PARTICIPATING GROUPS AND NUMERICAL APPROACHES

In Table 1 the different groups that provided results for the present benchmark computation are listed, and the individual test cases for which results were provided are also indicated. In Table 2 the numerical methods and implementations of the participating groups are summarized. Only the major features which are the most important for the evaluation of the results are given. The following abbreviations are used in the table: Finite difference method (FD), Finite volume method (FV), Finite element method (FE), Navier-Stokes equations (NS) and Multigrid method (MG). PEAK means the peak performance in MFlops and LINP the Linpack1000 MFlop-rate.

Table 1: Participating groups and test cases for which results were provided. The $p$ indicates that only parts of the required results for the corresponding test case were given, and $x$ indicates a full set of results

| Participants/test cases | 2D |  |  | 3D |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | 1 | 2 | 3 | 1Q | 1Z | 2Q |  | 3Q | 3Z |
| 1) RWTH Aachen, Aerodynamisches Institut E. Krause, M. Weimer, M. Meinke | x | x | x | x | x |  | p | p | p |
| 2) ASC GmbH (TASCflow) <br> F. Menter, G. Scheuerer |  |  |  |  | x |  |  |  |  |
| 3) TU Berlin Inst. für Strömungsmechanik F. Thiele, L. Xue | p | p | p | p | p | p | p | p | p |
| 4) TU Chemnitz, Fakultät für Mathematik A. Meyer, S. Meinel, U. Groh, M. Pester | x | x | x |  |  |  |  |  |  |
| 5) Daimler-Benz AG (STAR-CD) F. Klimetzek |  |  |  | p |  |  |  |  |  |
| 6) Univ. Duisburg, Inst. für Verbrennung und Gasdyn. D. Hänel, O. Filippova | x | x | p | p | p | p | p | p | p |
| 7) Univ. Erlangen, Lehrstuhl für Strömungsmechanik F. Durst, M. Schäfer, K. Wechsler | x | x | x | x | x |  | x |  | x |
| 8) Univ. Freiburg, Inst. für Angewandte Mathematik E. Bänsch, M. Schrul | x | x | x | x | x |  |  | p | p |
| 9) Univ. Hamburg, Inst. für Schiffbau M. Perić, S. Muzaferija, V. Seidl | x | x | x |  | x |  | p |  |  |
| 10) Univ. Heidelberg, Inst. für Angewandte Mathematik R. Rannacher, S. Turek | x | x | x | x | x | x | x | x | x |
| 11) TU Karlsruhe, Inst. für Hydromechanik <br> W. Rodi, M. Pourquie |  |  |  | x |  |  |  | p |  |
| 12) Univ. Karlsruhe, Inst. für Therm. Strömungsmasch. C.-H. Rexroth, S. Wittig | x |  |  |  |  |  |  |  |  |
| 13) Kyoto Inst. of Tech., Dept. of Mech. and Syst. Eng. N. Satofuka, H. Tokunaga, H. Hosomi | x | x |  |  |  |  |  |  |  |
| 14) Univ. Magdeburg, Inst. für Analysis und Numerik L. Tobiska, V. John, U. Risch, F. Schieweck | x |  | x |  |  |  |  |  |  |
| 15) TU München, Inst. für Informatik <br> C. Zenger, M. Griebel, R. Kreißl, M. Rykaschewski | x | x | x | x |  | p |  | p |  |
| 16) UBW München, Inst. f. Strömungsmech. u. Aerodyn. H. Wengle, M. Manhart |  |  |  | x |  | p |  |  |  |
| 17) Univ. Stuttgart, Inst. für Computeranwendungen G. Wittum, H. Rentz-Reichert | x |  |  |  |  |  |  |  |  |

Table 2: Numerical methods and implementation of participating groups

|  | Space discretization | Time discretization | Solver | Implementation |
| :---: | :--- | :--- | :--- | :--- |
| 1 | FD, blockstructured <br> non-staggered <br> QUICK upwinding | fully implicit 2nd ord. <br> equidistant | artificial compressibility <br> expl. 5-step Runge-Kutta <br> FAS-MG (steady) <br> line-Jacobi (unsteady) | serial <br> Fujitsu VPP500 <br> 1600 PEAK |
| 2 | FV, blockstructured <br> 2nd ord. upwindig | implicit Euler <br> equidistant | ILU with algebraic MG <br> for linear problems | serial <br> IBM RS6000/370 <br> 37 LINP |
| 3 | FV, blockstructured <br> non-staggered <br> QUICK upwinding | fully implicit 2nd ord. <br> equidistant | stream function form <br> fixed-point iteration <br> ILU for lin. subproblems | serial <br> SGI-Indigo2 <br> 75 PEAK |
| 4 | FE, blockstructured <br> 4 Q1-Q1 <br> BTD stabilisation | Projection 2 (Gresho) <br> Crank-Nicolson (diff.) <br> explicit Euler (conv.) <br> adaptive | pseudo time step (steady) <br> PCG for lin. subproblems <br> hierarch. preconditioning | parallel <br> GC/PP32 |
| 5a | FV, unstructured <br> 1st ord. upwind | STARCD software | pressure correction | Cray T3D/16 |

Table 2: (continued)

|  | Space discretization | Time discretization | Solver | Implementation |
| :---: | :---: | :---: | :---: | :---: |
| 10 | FE, blockstructured Q1(rot)-Q0 adaptive upwind | 2nd order fract. step projection method adaptive | fixed-point iteration MG for lin. NS with Vanka smoother (steady) MG for scalar lin. subproblems (unsteady) | serial IBM RS6000/590 90 LINP |
| 11 | FV, structured CDS with momentum interpolation | explicit <br> 3rd ord. Runge-Kutta equidistant | SIMPLE <br> ILU for lin. subprobl. | $\begin{aligned} & \text { serial } \\ & \text { SNI S600/20 } \\ & 5000 \text { PEAK } \end{aligned}$ |
| 12 | FV, unstructured non-staggered adapt. 2nd ord. DISC deferred correction | - | SIMPLEC ILU-BICGSTAB for lin. subproblems | serial <br> SUN SS10 <br> 5.5 LINP |
| 13a | FD, structured | explicit Euler equidistant | stream function form pseudo time step SOR for lin. subprobl. | serial <br> IBM RS6000/590 <br> 90 LINP |
| 13b | FD, structured | explicit 4th order Runge-Kutta-Gill equidistant | stream function form SOR for lin. subprobl. | serial <br> IBM RS6000/590 <br> 90 LINP |
| 14a | FE, blockstructured P1-P0 <br> (Crouzeix-Raviart) <br> 1st order upwind | - $\quad \square$ | nonlinear MG Vanka smoother | $\begin{aligned} & \text { serial } \\ & \text { HP737/125 } \\ & 6.6 \text { LINP } \end{aligned}$ |
| 14b | FE, blockstructured Q1(rot)-Q0 <br> 1st order upwind |  | fixed-point iteration MG for lin. NS with Vanka smoother | $\begin{aligned} & \text { parallel } \\ & \text { GC/PP96 } \\ & \text { 96x13.9 LINP } \end{aligned}$ |
| 14c | FE, unstructured P1-P0 (Cr.-Rav.) Samarskij upwind adaptive refinement | $\mathrm{BDF}(2)$, equidistant | fixed-point iteration GMRES for pressure Schur-complement lin. MG for velocity | $\begin{aligned} & \text { parallel } \\ & \text { GC/PP24 } \\ & 24 \times 13.9 \text { LINP } \end{aligned}$ |
| 15a | FD, structured staggered, orthogonal CDS/UDS flux-blend. | explicit Euler adaptive | SOR for pressure | serial <br> HP720 <br> 7.4 LINP |
| 15b | FD, structured staggered, orthogonal CDS/UDS flux blend. | explicit Euler adaptive | SOR for pressure | parallel <br> HP720 cluster <br> 8x7.4 LINP |
| 16 | FV, blockstructured CDS | explicit <br> 2nd ord. leap-frog time-lagged diff. | pressure correction Gauss-Seidel for lin. subproblems | serial <br> SGI Indigo <br> 9.6 LINP <br> Convex C3820 <br> 19.2 LINP |
| 17 | FV, unstructured adaptive upwind | - | fixed-point iteration MG for lin. NS BILU $_{\beta}$ smoother | $\begin{aligned} & \text { serial } \\ & \text { SGI R4400 } \\ & \text { 8.3 LINP } \end{aligned}$ |

## 4. RESULTS

The results of the benchmark computations are summarized in Tables 3-11. The number in the first column refers to the methods given in Table 2. The last column contains the performance of the computer used (as given by the contributors), either the Linpack1000 Mflop rate (LINP) or the peak performance (PEAK), which of course should be taken into account when comparing the different computing times. The column "unknowns" refers to the total number, i.e. the sum of unknowns for all velocity components and pressure. The CPU timings are all given in seconds. In the last row of each table estimated intervals for the "exact" results are indicated (as suggested by the authors on the basis of the obtained solutions).

We remark that for the 2D time-periodic test case 2D-2 also measurements were carried out, where the Strouhal number and time-averaged velocity profiles at different locations along the channel are determined experimentally. However, a direct comparison with the numerical results in Table 4 is problematic, because owing to the short distance between the inlet and the cylinder for the computations, the flow conditions in front of the cylinder are slightly different. To give some comparison with method 7a (see Table 2), a computation with a longer distance between the inlet and cylinder was carried out. The experimentally obtained Strouhal number of $S t=0.287 \pm 0.003$ agrees very well with the numerically computed value of $S t=0.289$. A comparison of time-averaged velocity profiles can be seen in Fig. 4, which also are in fairly good agreement.


Figure 4: Comparison of experimental and numerical time-averaged velocity profiles for test case 2D-2 with extended inlet part.

Table 3: Results for steady test case 2D-1

|  | Unknowns | $c_{D}$ | $c_{L}$ | $L_{a}$ | $\Delta P$ | Mem. | CPU time | MFlop rate |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 200607 | 5.5567 | 0.0106 | 0.0845 | 0.1172 | 15 | 788 | 1600 PEAK |
|  | 51159 | 5.5567 | 0.0106 | 0.0843 | 0.1172 | 4 | 273 |  |
|  | 13299 | 5.5661 | 0.0105 | 0.0835 | 0.1169 | 1 | 144 |  |
| 3a | 10800 | 5.6000 | 0.0120 | 0.0720 | 0.1180 | 2.5 | 121 | 75 PEAK |
| 4 | 297472 | 5.5678 | 0.0105 | 0.0847 | 0.1179 | 137 | 31000 | 445 LINP |
|  | 75008 | 5.5606 | 0.0107 | 0.0849 | 0.1184 | 73 | 8000 |  |
|  | 19008 | 5.5528 | 0.0118 | 0.0857 | 0.1199 | 57 | 2000 |  |
| 6 | 1314720 | 5.8190 | 0.0110 | 0.0870 | 0.1230 | 40 | 80374 | 13 LINP |
|  | 332640 | 5.7740 | 0.0030 | 0.0830 | 0.1230 | 10 | 10461 |  |
|  | 85140 | 5.7890 | -0.0060 | 0.0870 | 0.1230 | 2.6 | 1262 |  |
| 7a | 294912 | 5.5846 | 0.0106 | 0.0846 | 0.1176 | 75 | 192 | 13 LINP |
|  | 73728 | 5.5852 | 0.0105 | 0.0845 | 0.1176 | 19 | 47 |  |
|  | 18432 | 5.5755 | 0.0102 | 0.0842 | 0.1175 | 5 | 13 |  |
| 8a | 20487 | 5.5760 | 0.0110 | 0.0848 | 0.1170 | 9.0 | 2574 | 8.3 LINP |
|  | 6297 | 5.5710 | 0.0130 | 0.0846 | 0.1160 | 2.9 | 362 |  |
|  | 2298 | 5.4450 | 0.0200 | 0.0810 | 0.1110 | 1.3 | 109 |  |
| 9a | 240000 | 5.5803 | 0.0106 | 0.0847 | 0.1175 | 53 | 9200 | 34 LINP |
|  | 60000 | 5.5786 | 0.0106 | 0.0847 | 0.1173 | 10 | 1400 |  |
|  | 15000 | 5.5612 | 0.0109 | 0.0848 | 0.1166 | 2.5 | 200 |  |
| 10 | 2665728 | 5.5755 | 0.0106 | 0.0780 | 0.1173 | 350 | 677 | 90 LINP |
|  | 667264 | 5.5718 | 0.0105 | 0.0770 | 0.1169 | 89 | 169 |  |
|  | 167232 | 5.5657 | 0.0102 | 0.0730 | 0.1161 | 22 | 52 |  |
|  | 42016 | 5.5608 | 0.0091 | 0.0660 | 0.1139 | 5 | 18 |  |
| 12 | 32592 | 5.5069 | 0.0132 | 0.0830 | 0.1155 | 18 | 1796 | 5.5 LINP |
|  | 26970 | 5.5125 | 0.0056 | 0.0827 | 0.1154 | 15 | 1099 |  |
|  | 22212 | 5.6026 | -0.0031 | 0.0815 | 0.1167 | 13 | 3437 |  |
| 13a | 25410 | 5.6145 | 0.0159 | 0.8315 | 3.0002 | 4 | 14203 | 90 LINP |
|  | 12738 | 5.6114 | 0.0169 | 0.8224 | 2.9943 | 2 | 3018 |  |
|  | 6562 | 5.7377 | 0.0514 | 0.8107 | 3.2277 | 1 |  |  |
| 14a | 3077504 | 5.6323 | 0.0137 | 0.0782 | 0.1159 | 214 | 15300 | 6.6 LINP |
|  | 768704 | 5.6382 | 0.0102 | 0.0775 | 0.1156 | 53 | 5490 |  |
|  | 191840 | 5.5919 | -0.0009 | 0.0750 | 0.1143 | 13 | 2800 |  |
| 14b | 30775296 | 5.5902 | 0.0108 | 0.0853 | 0.1174 | 5340 | 1534 | 1334 LINP |
|  | 7695104 | 5.6010 | 0.0110 | 0.0844 | 0.1174 | 1341 | 400 |  |
|  | 1922432 | 5.6227 | 0.0113 | 0.0833 | 0.1172 | 338 | 119 |  |
| 14c | 797010 | 5.5708 | 0.0167 | 0.0837 | 0.1168 | 460 | 8000 | 334 LINP |
|  | 363457 | 5.5598 | 0.0142 | 0.0835 | 0.1166 | 230 | 3290 |  |
|  | 176396 | 5.5106 | 0.0046 | 0.0835 | 0.1150 | 110 | 2560 |  |
| 15a | 432960 | 5.5602 | 0.0329 | 0.0730 | 0.1054 | 4.4 | 179986 | 7.4 LINP |
|  | 108240 | 5.6300 | 0.0751 | 0.0720 | 0.1037 | 1.1 | 13593 |  |
|  | 27060 | 5.7769 | 0.2085 | 0.0680 | 0.0998 | 0.3 | 688 |  |
| 17 | 111342 | 5.5610 | 0.0107 |  | 0.1170 | 87 | 2568 | 8.3 LINP |
|  | 60804 | 5.5520 | 0.0102 |  | 0.1168 | 47 | 1092 |  |
|  | 19416 | 5.5160 | 0.0099 |  | 0.1158 | 15 | 373 |  |
|  | lower bound | 5.5700 | 0.0104 | 0.0842 | 0.1172 |  |  |  |
|  | upper bound | 5.5900 | 0.0110 | 0.0852 | 0.1176 |  |  |  |

Table 4: Results for time-periodic test case 2D-2

|  | Unknowns |  | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | St | $\Delta P$ | Mem. | CPU time | MFlop rate |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Space | Time |  |  |  |  |  |  |  |
| 1 | 267476 | 67 | 3.2224 | 0.9672 | 0.2995 | 2.4814 | - | - | 1600 PEAK |
|  | 267476 | 34 | 3.2030 | 0.9223 | 0.2941 | 2.4664 | - | - |  |
|  | 267476 | 18 | 3.1605 | 0.8026 | 0.2901 | 2.4466 | - | - |  |
|  | 68212 | 67 | 3.2171 | 0.9591 | 0.2995 | 2.5009 | - | - |  |
|  | 17732 | 68 | 3.2168 | 0.9295 | 0.2979 | 2.5573 | - | - |  |
| 3 | 12800 | 34 | 3.2200 | 0.9720 | 0.2960 | 2.4700 | 2.5 | 789 | 75 PEAK |
| 4 | 297472 | 670 | 3.2460 | 0.9840 | 0.2985 | 2.4900 | 137 | 6600 | 445 LINP |
|  | 297472 | 338 | 3.2710 | 0.9800 | 0.2959 | 2.4870 | 137 | 3400 |  |
|  | 297472 | 172 | 3.3200 | 0.9720 | 0.2907 | 2.4810 | 137 | 1700 |  |
|  | 75008 | 670 | 3.2410 | 0.9910 | 0.2985 | 2.5020 | 73 | 2350 |  |
|  | 19008 | 674 | 3.2320 | 1.0260 | 0.2967 | 2.5320 | 57 | 1350 |  |
| 6 | 332640 | 12000 | 4.1210 | 1.6120 | 0.3330 | 3.1420 | 10 | 10086 | 13 LINP |
|  | 85140 | 6000 | 4.7330 | 2.0600 | 0.3380 | 3.4300 | 2.6 | 1259 |  |
| 7 a | 294912 | 36 | 3.2358 | 1.0069 | 0.3003 | 2.4892 | 75 | 6167 | 13 LINP |
|  | 294912 | 19 | 3.2356 | 1.0000 | 0.2973 | 2.4871 | 75 | 6391 |  |
|  | 294912 | 10 | 3.2152 | 0.9028 | 0.2881 | 2.4715 | 75 | 4994 |  |
|  | 73728 | 36 | 3.2443 | 1.0261 | 0.2994 | 2.4929 | 19 | 1946 |  |
|  | 18432 | 36 | 3.2706 | 1.0695 | 0.2968 | 2.5035 | 5 | 445 |  |
| 8a | 29084 | 66 | 3.2240 | 1.0060 | 0.3020 | 2.4860 | 11 | 4992 | 8.3 LINP |
|  | 29084 | 33 | 3.2470 | 1.0740 | 0.3030 | 2.5010 | 11 | 3777 |  |
|  | 29084 | 16 | 3.2900 | 1.2500 | 0.3130 | 2.5700 | 11 | 3217 |  |
|  | 8764 | 66 | 3.1740 | 0.9640 | 0.3000 | 2.4630 | 3.6 | 1000 |  |
|  | 2978 | 70 | 2.8920 | 0.5540 | 0.2890 | 2.2870 | 1.5 | 339 |  |
| 9a | 240000 | 5000 | 3.2267 | 0.9862 | 0.3017 | 2.4833 | 53 | 32500 | 34 LINP |
|  | 60000 | 10000 | 3.2232 | 0.9830 | 0.3012 | 2.4773 | 10 | 8550 |  |
|  | 60000 | 5000 | 3.2232 | 0.9832 | 0.3012 | 2.4773 | 10 | 4500 |  |
|  | 60000 | 2500 | 3.2232 | 0.9836 | 0.3012 | 2.4773 | 10 | 3400 |  |
|  | 15000 | 5000 | 3.2058 | 0.9651 | 0.2994 | 2.4587 | 2.5 | 3240 |  |
| 10 | 667264 | 612 | 3.2314 | 0.9999 | 0.2973 | 2.4707 | 128 | 8545 | 90 LINP |
|  | 667264 | 204 | 3.2351 | 1.0123 | 0.2957 | 2.4734 | 128 | 2850 |  |
|  | 667264 | 68 | 3.2771 | 1.1205 | 0.2997 | 2.4961 | 128 | 1065 |  |
|  | 167232 | 188 | 3.2498 | 1.0081 | 0.2927 | 2.4410 | 32 | 655 |  |
|  | 42016 | 164 | 3.2970 | 0.8492 | 0.2713 | 2.3423 | 8 | 147 |  |
| 13b | 25410 | 6755 | 3.1822 | 1.0692 | 0.2960 | 2.6066 | 5.1 | 44710 | 90 LINP |
|  | 25410 | 3877 | 3.1895 | 1.0883 | 0.2968 | 2.6057 | 4.8 | 27175 |  |
|  | 25410 | 1678 | 3.2043 | 1.1268 | 0.2979 | 2.5307 | 4.7 | 13045 |  |
|  | 12738 | 6799 | 3.1945 | 1.1233 | 0.2941 | 2.6140 | 2.9 |  |  |
|  | 6562 | 7223 | 3.1317 | 1.2961 | 0.2768 | 3.0253 | 1.8 |  |  |
| 15a | 432960 | 7790 | 3.0804 | 0.7256 | 0.2778 | 2.1330 | 4.4 | 108844 | 7.4 LINP |
|  | 108240 | 4003 | 3.1677 | 0.6880 | 0.2646 | 2.0954 | 1.1 | 34876 |  |
|  | 108240 | 3859 | 3.1096 | 0.8249 | 0.2841 | 2.1105 | 1.1 | 58003 |  |
|  | 27060 | 1985 | 3.2544 | 0.5658 | 0.2336 | 1.9727 | 0.3 | 3796 |  |
|  | 27060 | 1670 | 3.1759 | 0.7656 | 0.2740 | 1.9961 | 0.3 | 4188 |  |
|  | lower bound upper bound |  | 3.2200 | 0.9900 | 0.2950 | 2.4600 |  |  |  |
|  |  |  | 3.2400 | 1.0100 | 0.3050 | 2.5000 |  |  |  |

Table 5: Results for unsteady test case 2D-3

|  | Unknowns |  | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | $\Delta P$ | Mem. | CPU time | MFlop rate |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Space | Time |  |  |  |  |  |  |
| 1 | 267476 | 400 | 2.9387 | 0.3504 | -0.1048 | - | - | 1600 PEAK |
|  | 68212 | 800 | 2.9459 | 0.4492 | -0.1057 | - | - |  |
|  | 17732 | 800 | 2.9532 | 0.3908 | -0.1007 |  |  |  |
| 3 | 12800 | 800 | 2.9600 | 0.4300 | -0.0976 | 2.5 | 7567 | 75 PEAK |
| 4 | 297472 | 16000 | 2.9715 | 0.4806 | -0.1101 | 137 | 160000 | 445 LINP |
|  | 297472 | 8000 | 2.9984 | 0.4794 | -0.1035 | 137 | 88000 |  |
|  | 297472 | 4000 | 3.0508 | 0.4750 | -0.1018 | 137 | 48000 |  |
|  | 75008 | 16000 | 2.9660 | 0.4903 | -0.1098 | 73 | 47000 |  |
|  | 19008 | 16000 | 2.9551 | 0.5228 | -0.1061 | 57 | 31000 |  |
| 6 | 332640 | 36960 | 3.8420 | 1.1100 | 0.0200 | 10 | 19772 | 13 LINP |
|  | 85140 | 9460 | 4.5310 | 1.7610 | 0.0090 | 2.6 | 2511 |  |
| 7 a | 294912 | 800 | 2.9520 | 0.4793 | -0.1086 | 75 | 43119 | 13 LINP |
|  | 294912 | 400 | 2.9520 | 0.4787 | -0.1016 | 75 | 29165 |  |
|  | 294912 | 200 | 2.9512 | 0.4021 | -0.1047 | 75 | 22141 |  |
|  | 73728 | 800 | 2.9511 | 0.4711 | -0.0995 | 19 | 10003 |  |
|  | 18432 | 800 | 2.9461 | 0.4638 | -0.1024 | 5 | 2847 |  |
| 8 a | 21508 | 1600 | 2.9200 | 0.4910 | -0.1110 | 8 | 44028 | 8.3 LINP |
|  | 21508 | 800 | 2.9210 | 0.5390 | -0.1140 | 8 | 31481 |  |
|  | 21508 | 400 | 2.9230 | 0.7250 | -0.1160 | 8 | 25512 |  |
|  | 5822 | 1600 | 2.8160 | 0.3560 | -0.1060 | 2.5 | 9294 |  |
|  | 1705 | 1600 | 2.7220 | 0.0055 | -0.1220 | 1.1 | 2109 |  |
| 9a | 240000 | 5000 | 2.9505 | 0.4539 | -0.1095 | 53 | 220000 | 34 LINP |
|  | 60000 | 10000 | 2.9483 | 0.4651 | -0.1062 | 10 | 92000 |  |
|  | 60000 | 5000 | 2.9483 | 0.4630 | -0.1062 | 10 | 64000 |  |
|  | 60000 | 2500 | 2.9482 | 0.4575 | -0.1039 | 10 | 38000 |  |
|  | 15000 | 5000 | 2.9397 | 0.4349 | -0.1095 | 2.5 | 22000 |  |
| 10 | 667264 | 4540 | 2.9538 | 0.4782 | -0.1053 | 128 | 62734 | 90 LINP |
|  | 667264 | 1612 | 2.9566 | 0.5533 | -0.1029 | 128 | 22431 |  |
|  | 667264 | 704 | 3.0650 | 0.8443 | -0.1090 | 128 | 11832 |  |
|  | 167232 | 4068 | 2.9776 | 0.4768 | -0.1097 | 32 | 14005 |  |
|  | 42016 | 2908 | 3.0949 | 0.3223 | -0.0951 | 8 | 2532 |  |
| 14c | 638880 | 800 | 3.0599 | 0.6326 | -0.1100 | 550 | 740000 | 334 LINP |
|  | 858848 | 800 | 3.1441 | 0.5266 | -0.1142 | 850 | 660000 | 668 LINP |
| 15a | 432960 | 9060 | 2.8916 | 0.2649 | -0.0987 | 4.4 | 237397 | 7.4 LINP |
|  | 108240 | 4070 | 2.8927 | 0.3171 | -0.0956 | 1.1 | 29140 |  |
|  | 108240 | 4020 | 3.0134 | 0.2921 | -0.0945 | 1.1 | 25697 |  |
|  | 27060 | 2857 | 3.1817 | 0.2702 | -0.1138 | 0.3 | 3371 |  |
|  | 27060 | 2013 | 3.0098 | 0.3973 | -0.0941 | 0.3 | 2541 |  |
|  | lower bound upper bound |  | 2.9300 | 0.4700 | -0.1150 |  |  |  |
|  |  |  | 2.9700 | 0.4900 | -0.1050 |  |  |  |

Table 6: Results for steady test case 3D-1Q

|  | Unknowns | $c_{D}$ | $c_{L}$ | $\Delta \mathrm{P}$ | Mem. | CPU time | MFlop rate |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 1 | 2530836 | 7.6415 | 0.0673 | 0.1740 | 251 | 1975 | 5000 PEAK |
|  | 657492 | 7.6029 | 0.0665 | 0.1738 | 72 | 702 | 1600 PEAK |
| 3 | 634872 | 7.6100 | 0.0642 | 0.1730 | 72 | 1935 | 1408 LINP |
| 5 a | 1472000 | 7.9200 | 0.0645 | 0.1751 | 121 | 127984 | 13 LINP |
|  | 184000 | 8.0400 | 0.0642 | 0.1722 | 17 | 3805 |  |
|  | 23000 | 7.6600 | 0.0720 | 0.1609 | 3 | 73 |  |
| 5 b | 1472000 | 7.4400 | 0.0615 | 0.1721 |  |  |  |
|  | 184000 | 7.2800 | 0.0582 | 0.1673 |  |  |  |
|  | 23000 | 6.7400 | 0.0615 | 0.1509 |  |  |  |
| 6 | 6303750 | 8.0930 | 0.0700 |  | 43 | 168657 | 13 LINP |
| 7 a | 454656 | 7.5395 | 0.0797 | 0.1715 | 115 | 9525 | 13 LINP |
|  | 56832 | 7.1280 | 0.0861 | 0.1616 | 13 | 1280 |  |
|  | 7104 | 6.4590 | 0.0988 | 0.1385 | 3 | 88 |  |
| 8 a | 362613 | 7.6480 | 0.0670 | 0.1751 | 126 | 46970 | 13.2 LINP |
|  | 73262 | 7.6530 | 0.0590 | 0.1766 | 28 | 6590 |  |
| 8 b | 97822 | 7.6340 | 0.0660 | 0.1742 | 38 | 8648 | 13.2 LINP |
| 10 | 6094976 | 7.6148 | 0.0600 | 0.1729 | 690 | 8244 | 90 LINP |
|  | 768544 | 7.5622 | 0.0503 | 0.1683 | 88 | 1267 |  |
|  | 97736 | 7.3069 | 0.0348 | 0.1590 | 10 | 380 |  |
| 11 | 1425600 | 7.7583 | 0.0511 | 0.1744 | 100 | 2538 | 5000 PEAK |
|  | 460800 | 7.7673 | 0.0406 | 0.1721 | 38 | 536 |  |
|  | 128000 | 7.2372 | 0.0602 | 0.1611 | 17 | 86 |  |
| 15 b | 6724000 | 6.0770 | 0.0859 | 0.0825 | 64 | 10600 | 52 LINP |
|  | 1681000 | 5.5060 | 0.1420 | 0.0796 | 16 | 1400 |  |
| 16 | 2007040 | 7.3700 | 0.0619 | 0.1720 | 25 | 45000 | 19.2 LINP |
|  | 405503 | 7.2500 | 0.0549 | 0.1680 | 6 | 11000 | 9.6 LINP |
|  | lower bound | 7.5000 | 0.0600 | 0.1720 |  |  |  |
|  | upper bound | 7.7000 | 0.0800 | 0.1800 |  |  |  |

Table 7: Results for steady test case 3D-1Z

|  | Unknowns | $c_{D}$ | $c_{L}$ | $\Delta \mathrm{P}$ | Mem. | CPU time | MFlop rate |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 1 | 2426292 | 6.1295 | 0.0093 | 0.1693 | 233 | 2097 | 5000 PEAK |
|  | 630564 | 6.1230 | 0.0095 | 0.1680 | 71 | 1238 | 1600 PEAK |
| 2 | 555000 | 6.1440 | 0.0074 | 0.1604 | 122 | 8731 | 26 LINP |
|  | 276800 | 5.8600 | 0.0042 | 0.1616 | 67 | 6094 |  |
| 3 | 608496 | 6.1600 | 0.0095 | 0.1690 | 74 | 4150 | 1408 LINP |
| 6 | 6303750 | 6.2330 | -0.0040 |  | 43 | 221706 | 13 LINP |
| 7 b | 12582912 | 6.1932 | 0.0093 | 0.1709 | 3571 | 2630 | 1779 LINP |
|  | 1572864 | 6.1868 | 0.0092 | 0.1703 | 518 | 1120 | 445 LINP |
|  | 196608 | 6.1366 | 0.0098 | 0.1673 | 71 | 460 | 111 LINP |
| 8a | 362613 | 6.1430 | 0.0084 | 0.1694 | 126 | 51280 | 13.2 LINP |
|  | 73262 | 6.0990 | 0.0067 | 0.1695 | 28 | 7178 |  |
| 9 | 2355712 | 6.1800 | -0.0010 | 0.1691 |  | 62000 | 90 LINP |
|  | 753664 | 6.1720 | 0.0090 | 0.1680 |  | 6000 |  |
|  | 94208 | 6.1310 | 0.0100 | 0.1605 |  | 950 |  |
| 10 | 6116608 | 6.1043 | 0.0079 | 0.1672 | 700 | 8440 | 90 LINP |
|  | 771392 | 5.9731 | 0.0059 | 0.1605 | 89 | 1466 |  |
|  | 98128 | 5.8431 | 0.0061 | 0.1482 | 11 | 290 |  |
|  | lower bound | 6.0500 | 0.0080 | 0.1650 |  |  |  |
|  | upper bound | 6.2500 | 0.0100 | 0.1750 |  |  |  |

Table 8: Results for time-periodic test case 3D-2Q

|  | Unknowns |  |  |  |  |  |  |  |
| :--- | ---: | ---: | :--- | ---: | ---: | ---: | ---: | ---: |
|  | Space | Time | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | $S t$ | Mem. | CPU time | MFlop rate |
| 3 | 634872 | 188 | 4.3170 | 0.0495 | 0.3130 | 74 | 3368 | 1408 LINP |
|  | 634872 | 95 | 4.3170 | 0.0495 | 0.3210 | 74 | 2754 |  |
| 6 | 6303750 | 18000 | 4.5870 | -0.0050 | - | 43 | 168657 | 13 LINP |
| 10 | 6094976 | 142 | 4.3923 | 0.0146 | 0.2777 | 840 | 29428 | 90 LINP |
|  | 6094976 | 124 | 4.3932 | 0.0191 | 0.2806 | 840 | 29945 |  |
|  | 6094976 | 84 | 4.4071 | 0.0896 | 0.2400 | 840 | 30372 |  |
|  | 768544 | - | 4.4819 | 0.0036 | - | 105 | - |  |
|  | 97736 | - | 4.5529 | -0.0080 | - | 13 | - |  |
| 16 | 2007040 | 1726 | 4.6738 | 0.0389 | 0.3488 | 25 | 20040 | 19.2 LINP |
|  | 405503 | 833 | 4.8808 | 0.0392 | 0.3610 | 6 | 10020 | 9.6 LINP |
|  | lower bound | $?$ | $?$ | $?$ |  |  |  |  |
|  | upper bound |  |  |  |  |  |  |  |

Table 9: Results for time-periodic test case 3D-2Z

|  | Unknowns |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Space | Time | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | $S t$ | Mem. | CPU time | MFlop rate |  |  |  |  |  |  |  |
| 1 | 630564 | 177 | 3.3018 | -0.0014 | 0.3390 | 78 | 26115 | 1600 PEAK |  |  |  |  |  |  |  |
| 3 | 608496 | - | 3.2250 | -0.0142 | - | 74 |  | 1408 LINP |  |  |  |  |  |  |  |
|  | 608496 | - | 3.2250 | -0.0142 | - |  |  |  |  |  |  |  |  |  |  |
| 6 | 6303750 | 18000 | 3.7920 | -0.0210 | - | 43 | 142646 | 13 LINP |  |  |  |  |  |  |  |
| 7 b | 12582912 | 93 | 3.3052 | -0.0105 | 0.3409 | 3571 | 24459 | 1779 LINP |  |  |  |  |  |  |  |
|  | 1572864 | 378 | 3.3057 | -0.0118 | 0.3172 | 518 | 9487 | 445 LINP |  |  |  |  |  |  |  |
|  | 1572864 | 261 | 3.3054 | -0.0118 | 0.2250 | 518 | 2740 | 445 LINP |  |  |  |  |  |  |  |
|  | 1572864 | 126 | 3.3050 | -0.0018 | 0.2400 | 518 | 1956 | 445 LINP |  |  |  |  |  |  |  |
|  | 196608 | - | 3.3121 | -0.0150 | - | 71 | - | 111 LINP |  |  |  |  |  |  |  |
| 9 b | 2355712 |  | 3.2968 |  |  |  |  | 90 LINP |  |  |  |  |  |  |  |
|  | 753664 |  | 3.3254 |  |  |  |  |  |  |  |  |  |  |  |  |
|  | 94208 |  | 3.3284 |  |  |  |  |  |  |  |  |  |  |  |  |
| 10 | 6116608 | 128 | 3.2950 | -0.0081 | 0.2912 | 840 | 31145 | 90 LINP |  |  |  |  |  |  |  |
|  | 6116608 | 120 | 3.2970 | -0.0025 | 0.2830 | 840 | 31730 |  |  |  |  |  |  |  |  |
|  | 6116608 | 80 | 3.3200 | 0.0480 | 0.2684 | 840 | 21586 |  |  |  |  |  |  |  |  |
|  | 771392 | 68 | 3.3801 | 0.0086 | 0.2343 | 105 | 2163 |  |  |  |  |  |  |  |  |
|  | 98128 | - | 3.4593 | -0.0102 | - | 13 |  | - |  |  |  |  |  |  |  |
|  | lower bound |  |  |  |  |  |  |  |  | 3.2900 | -0.0110 | 0.2900 |  |  |  |
|  | upper bound | 3.3100 | -0.0080 | 0.3500 |  |  |  |  |  |  |  |  |  |  |  |

Table 10: Results for unsteady test case 3D-3Q

|  | Unknowns |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | Space | Time | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | $\Delta P$ | Mem. | CPU time | MFlop rate |
| 1 | 657492 | 800 | 4.3804 | 0.0308 | -0.1392 | 78 | 121960 | 1600 PEAK |
| 3 | 634872 | 1600 | 4.3030 | 0.0476 | -0.1361 | 74 | 51253 | 1408 LINP |
|  | 634872 | 800 | 4.3020 | 0.0473 | -0.1354 | 74 | 37241 |  |
| 6 | 6303750 | 18000 | 4.8680 | 0.0310 |  | 43 | 168657 | 13 LINP |
| 8a | 362613 | 1000 | 4.5530 | 0.0137 | -0.1436 | 126 | 398000 | 58 LINP |
| 8b | 228451 | 1000 | 4.5080 | 0.0432 | -0.1427 | 105 | 915000 | 13.2 LINP |
| 10 | 6094976 | 772 | 4.4086 | 0.0133 | -0.1264 | 840 | 164749 | 90 LINP |
|  | 6094976 | 392 | 4.5698 | 0.0262 | -0.1213 | 840 | 89679 |  |
|  | 6094976 | 82 | 5.5709 | 0.1230 | 0.0183 | 840 | 35600 |  |
|  | 768544 | 696 | 4.5223 | 0.0061 | -0.1113 | 105 | 22747 |  |
|  | 97736 | 588 | 4.5820 | 0.0033 | -0.0718 | 13 | 3031 |  |
| 11 | 3712800 | 7720 | 4.3400 | 0.0500 | -0.0810 | 105 | 5711 | 5000 PEAK |
|  | 1523200 | 7720 | 4.4000 | 0.0480 | -0.1160 | 48 | 2741 |  |
|  | 352000 | 7720 | 4.3600 | 0.0680 | -0.1090 | 18 | 706 |  |
|  | lower bound |  |  |  |  |  |  |  |
|  | 4.3000 | 0.0100 | -0.1400 |  |  |  |  |  |
|  | upper bound | 4.5000 | 0.0500 | -0.1200 |  |  |  |  |

Table 11: Results for unsteady test case 3D-3Z

|  | Unknowns |  | $c_{\text {Dmax }}$ | $c_{\text {Lmax }}$ | $\Delta \mathrm{P}$ | Mem. | CPU time | MFlop rate |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Space | Time |  |  |  |  |  |  |
| 1 | 630564 | 800 | 3.2826 | 0.0027 | -0.1117 | 79 | 156460 | 1600 PEAK |
| 3 | 608496 | 1600 | 3.2590 | 0.0026 | -0.1072 | 74 | 76142 | 1408 LINP |
|  | 608496 | 800 | 3.2590 | 0.0026 | -0.1157 | 74 | 50764 |  |
| 6 | 6303750 | 18000 | 4.1600 | 0.0200 |  | 43 | 142646 | 13 LINP |
| 7b | 1572864 | 1600 | 3.3011 | 0.0026 | -0.1102 | 518 | 149923 | 445 LINP |
|  | 1572864 | 800 | 3.3008 | 0.0026 | -0.1105 | 518 | 93055 | 445 LINP |
|  | 1572864 | 400 | 3.3006 | 0.0026 | -0.1107 | 518 | 62026 | 445 LINP |
|  | 196608 | 1600 | 3.3053 | 0.0028 | -0.1066 | 71 | 63057 | 111 LINP |
| 8a | 362613 | 1000 | 3.2340 | 0.0028 | -0.1114 | 126 | 347000 | 58 LINP |
| 8b | 199802 | 1000 | 3.2120 | 0.0122 | -0.1112 | 105 | 846000 | 13.2 LINP |
|  | 98637 | 1000 | 3.2350 | 0.0123 | -0.1114 | 39 | 243000 |  |
| 10 | 6116608 | 668 | 3.2802 | 0.0034 | -0.0959 | 840 | 164837 | 90 LINP |
|  | 6116608 | 272 | 3.3748 | 0.0360 | -0.0603 | 840 | 77538 |  |
|  | 6116608 | 60 | 2.7312 | 0.0069 | -0.0682 | 840 | 29742 |  |
|  | 771392 | 724 | 3.3323 | 0.0033 | -0.0766 | 105 | 24745 |  |
|  | 98128 | 660 | 3.4200 | 0.0040 | -0.0407 | 13 | 5687 |  |
|  | lower bound upper bound |  | 3.2000 | 0.0020 | -0.0900 |  |  |  |
|  |  |  | 3.3000 | 0.0040 | -0.1100 |  |  |  |

## 5. DISCUSSION OF RESULTS

On the basis of the results obtained by these benchmark computations some conclusions can be drawn. These have to be considered with care, as the provided results depend on parameters which are not available for the authors of this report, e.g., design of the grids, setting of stopping criteria, quality of implementation, etc.

For five of the ten questions above the answers seem to be clear:

1. In order to compute incompressible flows of the present type (laminar) accurately and efficiently, one should use implicit methods. The step size restriction enforced by explicit time stepping can render this approach highly inefficient, as the physical time scale may be much larger than the maximum possible time step in the explicit algorithm. This is obvious from the results for the stationary cases in 2D and 3D, and also for the nonstationary cases in 2D. For the nonstationary cases in 3D only too few results on apparently too coarse meshes have been provided, in order to draw clear conclusions. This question requires further investigation.
2. Flow solvers based on conventional iterative methods on the linear subproblems have on fine enough grids no chance against those employing suitable multigrid techniques. The use of multigrid can allow computations on workstations (provided the problem fits into the RAM) for which otherwise supercomputers would have to be used. In the submitted solutions supercomputers (Fujitsu, SNI, CRAY) have mainly been used for their high CPU power but not for their large storage capacities. For example, in test case 3D-3Z
(Table 11) the solutions 1 and 3 require with about 600,000 unknowns on supercomputers significantly more CPU time than the solution 10 with the same number of unknowns on a workstation.
3. The most efficient and accurate solutions are based either on finite element or finite volume discretizations on contour adapted grids.
4. The computation of steady solutions by pseudo time-stepping techniques is inefficient compared with using directly a quasi-Newton iteration as stationary solver.
5. For computing sensitive quantities such as drag and lift coefficients, higher order treatment of the convective term is indispensable. The use of only first order upwinding (or crude approximation of curved boundaries) does not lead to satisfactory accuracy even on very fine meshes (several million unknowns in 2 D ).

For the remaining five questions the answers are not so clear. More test calculations will be necessary to reach more decisive conclusions. The following preliminary interpretations of the results obtained so far may become the subject of further discussion:
6. In computing nonstationary solutions, the use of operator splitting (pressure correction) schemes tends to be superior to the more expensive fully coupled approach, but this may depend on the problem as well as the quantity to be calculated (compare, e.g., for the test case 2D-3 (Table 5), the solution 14c with 7a and 10). Further, as fully coupled methods also use iterative correction within each time step (possibly adaptively controlled), the distinction between fully coupled and operator splitting approach is not so clear.
7. The use of higher than second-order discretizations in space appears promising with respect to accuracy, but there remains the question of how to solve efficiently the resulting algebraic problems (see the results of 8 for all test cases). The results provided for this benchmark are too sparse to allow a definite answer.
8. The most efficient solutions in this benchmark have been obtained on blockwise structured grids which are particularly suited for multigrid algorithms. There is no indication that fully unstructured grids might be superior for this type of problem, particularly with respect to solution efficiency (compare the CPU times reported for the solutions 7 and 9 in 2D). The winners may be hierarchically structured grids which allow local adaptive mesh refinement together with optimal multigrid solution.
9. From the contributed solutions to this benchmark there is no indication that aposteriori grid adaptation in space is superior to good hand-made grids (see the results of 14c). This, however, may drastically change in the future, particularly in 3D. Intensive development in this direction is currently in progress.
For nonstationary calculations, adaptive time step selection is advisible in order to achieve reliability and efficiency (see the results of 10).
10. The treatment of the nonlinearity by nonlinear multigrid has no clear advantage over the quasi-Newton iteration with multigrid for the linear subproblems (compare the results of 7 with those of 10 ). Again, it is the extensive use of well-tuned multigrid (wherever in the algorithm) which is decisive for the overall efficiency of the method.

## 6. CONCLUDING REMARKS

The authors would like to add some final remarks to the report presented. Although, this benchmark has been fairly successful as it has made possible some solidly based comparison between various solution approaches, it still needs further development. Particularly the following points are to be considered:

1. In the case 3D-3Z it should be the maximum absolute value of the lift which has to be computed as $c_{L}$ may become negative.
2. In the nonstationary test cases further characteristic quantities (e.g., time averages, pressure values, etc.) should be computed, as in some cases, by chance, "maximum values" may be obtained with good accuracy even without capturing the general pattern of the flow at all.
3. For the nonstationary 3D problems a higher Reynolds number should be considered, since in the present case $(\operatorname{Re}=100)$ the problem may be particularly hard as the flow tends to become almost stationary.

Even in the laminar case, the chosen nonstationary 3D problems showed to be harder than expected. In particular, it was apparently not possible to achieve reliable reference solutions for the test cases 3D-2Q and 3D-2Z. Hence the benchmark has to be considered as still open and everybody is invited to try again.

## ACKNOWLEDGMENTS

The authors thank all members of the various groups who have contributed results for the benchmark computations, K. Wechsler for his help in evaluating the results and J. Jovanovic and M. Fischer for carrying out the experiments.

