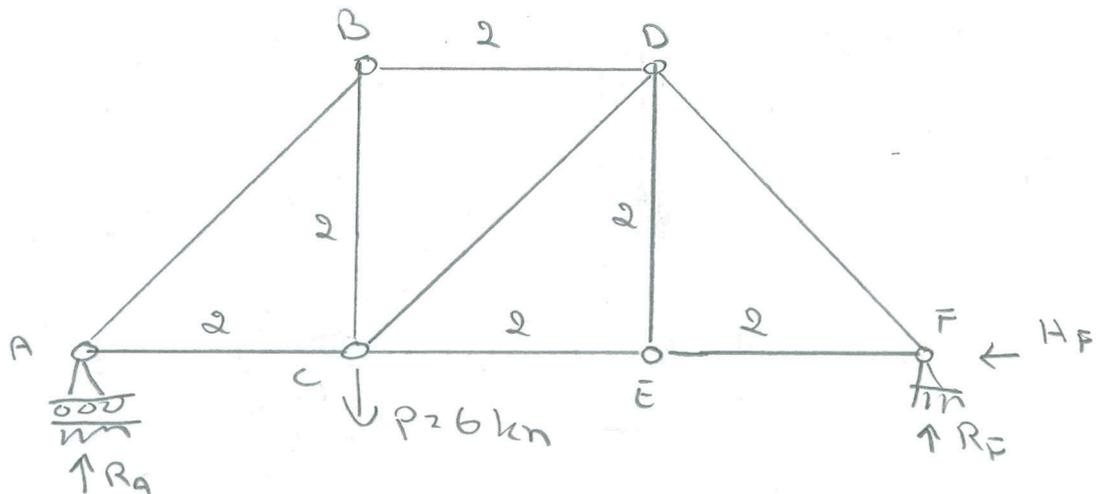


Oppgave 1



a) Statisk bestemthet kan bestemmes fra (for plane fagverk)

$$m + r = 2j$$

der

m - antall staver

r - antall reaksjons-/opplagerkrefter

j - antall knutepunkter

I dette tilfellet er

$$m = 9$$

$$r = 3 \quad (R_A, R_F, H_F)$$

$$2j = 2 \cdot 6$$

}
}
}

$$12$$

erlike

$$12$$

\Rightarrow STATISK
BESTEMT

b) Reaktionskraftene:

2

Fra likevekt:

$$\sum M_F = 0 \Rightarrow R_A \cdot 6 - P \cdot 4 = 0 \Rightarrow R_A = \frac{4}{6} P = \underline{\underline{4 \text{ kN}}}$$

$$\sum F_x = 0 \Rightarrow H_F = \underline{\underline{0}}$$

$$\sum F_y = 0 \Rightarrow R_A + R_F - P = 0 \Rightarrow R_F = P - R_A = \underline{\underline{2 \text{ kN}}}$$

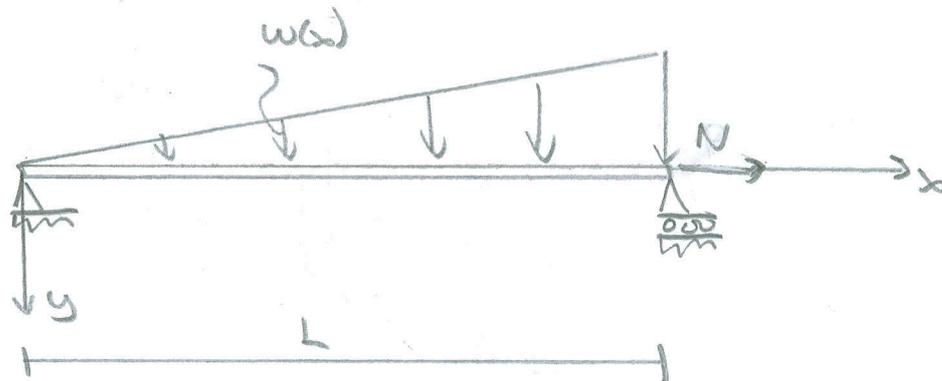
c) Stavkraftene i punkt A:



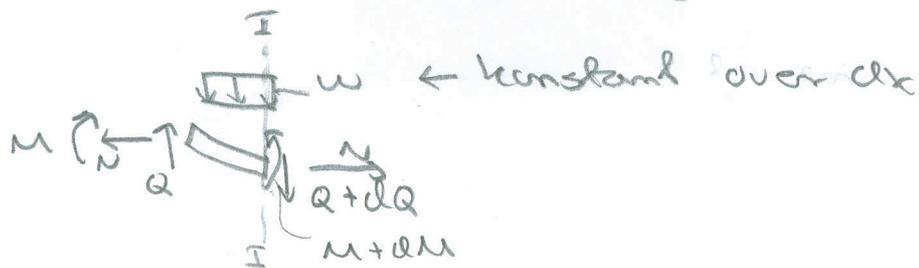
$$\begin{aligned} \sum F_y = 0 \quad F_{AB} \sin 45^\circ + R_A = 0 &\Rightarrow F_{AB} = -\frac{R_A}{\sin 45^\circ} \\ &= -\frac{4}{\frac{1}{2}\sqrt{2}} = -\frac{8}{\sqrt{2}} \\ &= \underline{\underline{-5,66 \text{ kN}}} \\ &\text{(trykke)} \end{aligned}$$

$$\begin{aligned} \sum F_x = 0 \quad F_{AB} \cos 45^\circ + F_{AC} = 0 \\ \Rightarrow F_{AC} = -F_{AB} \cos 45^\circ = \frac{8}{\sqrt{2}} \cdot \frac{1}{2}\sqrt{2} = \underline{\underline{4 \text{ kN}}} \\ \text{(strecke)} \end{aligned}$$

Oppgave 2



- a) På et infinitesimalt element av bjelken virker følgende krefter



Pga aksiallasten, stiller vi opp likevækt for deformert konstruksjon

$$\sum F_x = 0 \Rightarrow N - N = 0 \quad (1)$$

$$\begin{aligned} \sum F_y = 0 &\Rightarrow Q + dQ - Q + w dx = 0 \\ &\Rightarrow \frac{dQ}{dx} = -w \quad (2) \end{aligned}$$

$$\begin{aligned} \sum M_I = 0 &\Rightarrow -Q dx - M + \underbrace{w dx \frac{dx}{2}}_{\text{negligeres}} + M + dM + N dv = 0 \\ \underline{\underline{\frac{dM}{dx} = Q - N \frac{dv}{dx}}} &\quad (3) \end{aligned}$$

b) Fra (1) følger det at medlem to snelt kan uttrykkes som

$$Q = Q_0 - \int_0^x \frac{qx}{L} dx = Q_0 - \underline{\frac{qx^2}{2L}}$$

Fra (3), gitt at $N=0$, følger det at

$$\begin{aligned} M &= M_0 + \int_0^x Q dx = M_0 + \int_0^x \left(Q_0 - \frac{qx^2}{2L} \right) dx \\ &= \underline{M_0 + Q_0 x - \frac{qx^3}{6L}} \end{aligned}$$

Bestemmer Q_0 og M_0 fra randbetingelser:

$$1) M=0 \text{ for } x=0 \Rightarrow \underline{M_0=0}$$

$$2) M=0 \text{ for } x=L \Rightarrow \underline{Q_0 = \frac{qL}{6}}$$

Det gir

$$Q = \frac{qL}{6} - \frac{qx^2}{2L} \quad (4)$$

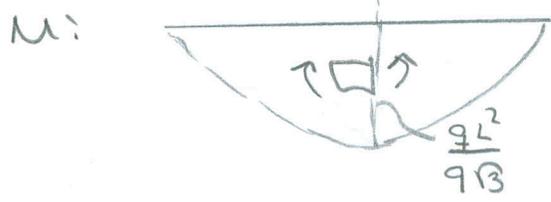
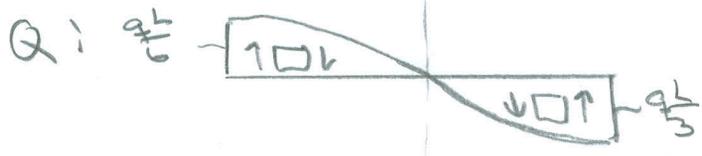
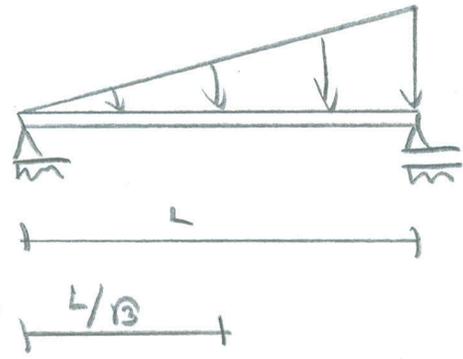
$$M = \frac{qL}{6} x - \frac{qx^3}{6L} \quad (5)$$

$$\text{Posisjonen hvor } Q=0: \frac{qL}{6} = \frac{qx^2}{2L} \Rightarrow x = \underline{\underline{\frac{L}{\sqrt{3}}}}$$

Siden $\frac{dM}{dx} = Q = 0$ ved $x = \frac{L}{\sqrt{3}}$ er dette mulig maksimum for M .

$$M_{\text{maks}} = \frac{qL}{6} \left(\frac{L}{\sqrt{3}} \right) - \frac{q}{6L} \left(\frac{L}{\sqrt{3}} \right)^3 = \underline{\underline{\frac{qL^2}{9\sqrt{3}}}}$$

c) Skjærkraft- og momentdiagram



d) Differensiallikningen

$$\frac{d^2 v}{dx^2} = -\frac{M}{EI} \quad (6)$$

Randbetingelsene:

$$I) \quad x=0: v=0$$

$$II) \quad x=L: v=0$$

e) Nedbøyningen av bjelken som funksjon av x

Fra (4) har vi momentfordelingen

Innsatt i (6) gir

$$EI \frac{d^2 v}{dx^2} = -M(x) = \frac{q}{6L} x^3 - \frac{qL}{6} x$$

Integrerer to ganger

$$EI \frac{dv}{dx} = \frac{q}{24L} x^4 - \frac{qL}{12} x^2 + C$$

$$EI v = \frac{q}{120L} x^5 - \frac{qL}{36} x^3 + Cx + D$$

Benytt de randbetingelsene til å bestemme C og D.

I) gir $D = 0$

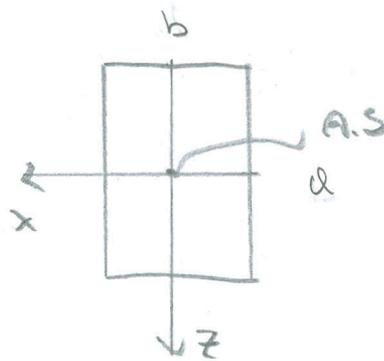
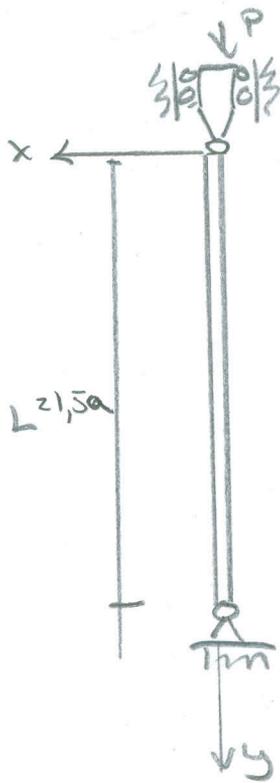
II) $C L = \frac{qL}{36} L^3 - \frac{q}{120L} L^5$

$$C = \frac{7}{360} q L^3$$

Derfor gir

$$v(x) = \frac{q}{120L} x^5 - \frac{qL}{36} x^3 + \frac{7}{360} q L^3 x$$

Oppgave 3



a) Knekklasten for en ledet søyle

$$P_c = \frac{\pi^2 EI}{L^2} = \frac{\pi^2 EI}{\left(\frac{3}{2}a\right)^2} = \frac{\pi^2 EI}{\frac{9}{4}a^2} = \underline{\underline{\frac{4\pi^2 EI}{9a^2}}}$$

b) Gitt fasongen på tverrsnittet er

$I_x > I_z$ og søylen vil knekke i planet (x-y)

$$I_z = \int_A x^2 dA = \underline{\underline{\frac{1}{12} bd^3}}$$

Kritisk trykspenning:

$$\begin{aligned} \sigma_c &= \frac{P_c}{A} = \frac{\frac{4\pi^2 EI_z}{9a^2}}{bd} = \frac{4\pi^2 EI_z}{9a^2 bd} = \frac{4\pi^2 E \frac{bd^3}{12}}{9a^2 bd} \\ &= \frac{\pi^2 E bd^3}{27a^2 bd} = \underline{\underline{\frac{\pi^2 E d^2}{27a^2}}} \end{aligned}$$